

# NOTES ON BASIC TURBULENCE

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PHYS 235 Course

"When I meet God, I'm gonna ask him two questions. Why Relativity? And why Turbulence? I believe he will have an answer for the first."

- Werner Heisenberg

Summary:

In this set of notes, basics of 3D, 2D and pipe flow turbulence have been discussed. States dominated by large spectral fluxes, correspond to turbulent cascades, in which nonlinear interactions couple source and sink at different scales by a sequence of local transfer events. Famous Kolmogorov cascade, as well as, 3D forward cascade and 2D inverse cascade has been discussed. Prandtl theory for pipe flows and similarity and differences and relevancy to fusion also discussed

## \* Basic Turbulence:

⇒ Considering Navier-Stokes Momentum Eqn:

$$\frac{\partial v}{\partial t} + v \cdot \nabla v + \nu \nabla^2 v = -\nabla p / \rho + \hat{F}_{ext}$$

$$\Rightarrow \frac{\partial}{\partial t} \langle \hat{v}^2 \rangle + \langle v \cdot \nabla \cdot v \rangle + \underbrace{\nu \langle (\nabla v)^2 \rangle}_{\text{viscous dissipation}} = \underbrace{\langle \hat{F} \cdot v \rangle}_{\text{energy input}}$$

⇒ Anal dissipative Force and Power are:

$$F_d \sim \rho A v^2 C_D(Re) \Rightarrow \frac{P_d}{M} \sim \frac{F_d v}{M} \sim C_D(Re) \frac{v^3}{l} \sim Re^0$$

- Macro-scale:  $P_d$  independent of  $Re$
- Micro-scale:  $\langle \hat{F} \cdot v \rangle \sim \nu \langle (\nabla v)^2 \rangle$   
 $\sim \frac{1}{r}$  singular flow shear

The question of turbulent cascades:

- How to Form singular  $\nabla v$
- Observed inertial range

Different turbulent structures have been discussed:

- k41 (Kolmogorov 41)
- 2D Turbulence
- Pipe Flow

Assuming Incompressibility  
 $\nabla \cdot v = 0$

Note that:  
 $v \cdot \nabla v = \nabla \left( \frac{v^2}{2} \right) - v \times \omega$

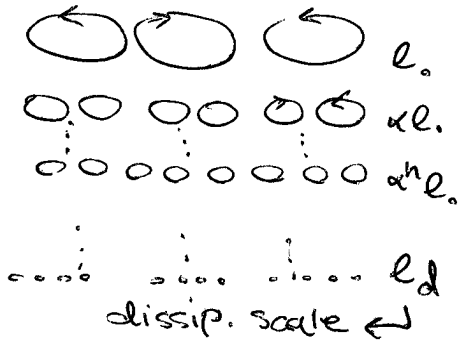
Reynolds Number  
 $Re \sim \frac{v(l)}{\nu}$

# Energy Flow in scale space $\Leftrightarrow$ K41 Cascade

- Assumptions for Kolmogorov's theory of High Reynolds number turbulence:

- Spatial Homogeneity
- Isotropy
- Self-similarity
- locality of interaction

↳ large scales can't distort/destroy small scales.



Constant Energy throughput for all inertial range scales:

$$\epsilon = \frac{v_0^3}{l_0} \sim \frac{v(l)^3}{l} \Rightarrow v(l) \sim \epsilon^{1/3} l^{1/3}$$

Data structure of flow field.

$$v(l) = |V(r+l) - V(r)| \text{ (rough in space)}$$

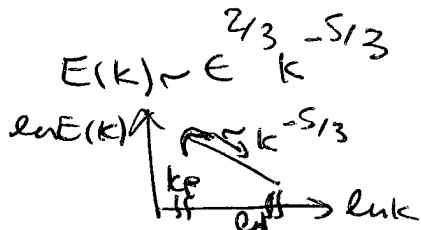
$$\frac{|V(r+l) - V(r)|}{|l|} \sim \frac{\epsilon^{1/3}}{l^{2/3}} \rightarrow \begin{cases} -\nabla v \text{ diverges as } l \rightarrow 0 \\ -\text{cutoff at dissipation scale} \end{cases}$$

⇒ Richardson Theory of particle separation:

How the distance between two particles grows in time in a turbulent flow.

$$\frac{dl}{dt} = v(l) \sim \epsilon^{1/3} l^{1/3} \Rightarrow \frac{dl}{l^{1/3}} = \epsilon^{1/3} dt \Rightarrow \boxed{l^2 \sim \epsilon^{2/3} t^3}$$

Richardson  
(Relative Separation)



Also Random-Walk

is rough (non-differentiable)

$$\delta x^2 \sim t$$

$$\frac{d}{dt} \delta x \sim l^{1/3}$$

Richardson Separation growth vs. Brownian Motion  
 $l \sim t^{3/2}$        $l \sim t^{1/2}$

↳ Larger eddies support larger speed, separation process is self-accelerating.

Review Article  
 [Fakovich, Gnedin, Vergassola, 2001]

To find dissipation scale, we balance the eddy straining rate  $\frac{|v(r+t) - v(r)|}{l} \sim \frac{\epsilon^{1/3}}{l^{2/3}}$

with viscous dissipation rate  $\nu/l^2$

$$\frac{1}{\tau(r)} \sim \frac{\nu}{l^2} \Rightarrow \frac{\epsilon^{1/3}}{l^{2/3}} \sim \frac{\nu}{l^2} \Rightarrow l = \frac{\nu^{3/4}}{\epsilon^{1/4}}$$

↳ decay rate      ↳ dissipation rate      Kolmogorov Microscale

Integral Scale (Outer scale)  
 less than this no longer eddy flows be carried  
 $l \sim \int \frac{dr v(x)v(x+r)}{|v(x)|^2}$

Why does energy cascade self-similarity to small scales?

Energy  $E = 1/2 \rho \langle v^2 \rangle$   
 Enstrophy  $\mathcal{E} = \langle \omega^2 \rangle$   
 vorticity  $\omega = \nabla \times v$

Vorticity + Vortex Dynamics  
 ↳ Enstrophy

3D Turbulence  
 ↳ Enstrophy generation  
 2D Turbulence  
 ↳ Enstrophy conservation

$$\frac{\partial v}{\partial t} + v \cdot \nabla v = -\frac{\nabla p}{\rho} + \nu \nabla^2 v$$

$$\frac{\partial v}{\partial t} = -\nabla \left( \frac{p}{\rho} + \frac{v^2}{2} \right) + \nu \nabla^2 v + v \times \omega$$

$$\frac{\partial \omega}{\partial t} = \underbrace{\nabla \times (v \times \omega)}_{-v \cdot \nabla \omega + \omega \cdot \nabla v} + \nu \nabla^2 \omega$$

$$\Rightarrow \frac{d\omega}{dt} - \nu \nabla^2 \omega = \omega \cdot \nabla \omega$$

look alike  $\frac{\partial B}{\partial t} - \mu \nabla^2 B = B \cdot \nabla v$

$$\frac{d}{dt} \langle w^2 \rangle = \langle w \cdot (w \cdot \nabla v) \rangle - \nu \langle (\nabla w)^2 \rangle$$

Enstrophy Production

How to estimate it?

$$\frac{d}{dt} \langle w^2 \rangle \sim \langle w^3 \rangle - \nu \langle (\nabla w)^2 \rangle$$

↳ signal of explosive growth

Finite time singularity!

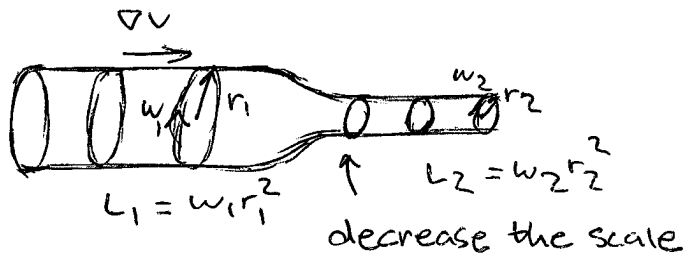
+

Vortex Tube Stretching

Finite time singularity of Enstrophy?

↓

Clay's Prize!



Idea:

⇒ 2D Turbulence:

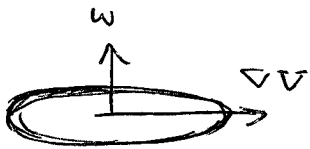
Maximum Enstrophy?

Minimum Enstrophy?

- Is very important paradigm for plasma turbulent.

- Vortex tube stretching doesn't exist here

( $w \cdot \nabla v = 0$ ) ⇒ Vorticity conserved locally



$$\frac{d}{dt} w + v \cdot \nabla w - \nu \nabla^2 w = 0$$

or by representing  $v$  using a stream

function ( $v = \nabla \varphi \times \hat{z}$ ):

Hasegawa-Wakatani

$$\nabla \cdot j = 0; \frac{d}{dt} n = 0$$

$$\partial_t \nabla^2 \varphi + v \cdot \nabla \nabla^2 \varphi + \nabla \cdot \nabla \varphi = C_D(n, \varphi) \leftarrow \text{resemble} \quad \partial_t \nabla^2 \varphi + v \cdot \nabla \nabla^2 \varphi - \nu \nabla^2 \nabla^2 \varphi = 0$$

$$\partial_t n + v \cdot \nabla n - D \nabla^2 n = C_D(n, \varphi) \leftarrow \text{limits of HW}$$

$$C = v_{th}^2 / \nu_{ee}$$

Hence in 2D Turbulence:

- Modulos  $U, E, \Omega$  are conserved.
- No vortex tube stretching!
- Vortex Merger  $\rightarrow$  like Biot-Savart Law

Energy:

$$E = \iint d^2x \frac{U^2}{2} = \iint d^2x \frac{|\nabla\phi|^2}{2}$$

Eustrophy:

$$\Omega = \iint d^2x \frac{\omega^2}{2} = \iint d^2x \frac{|\nabla^2\phi|^2}{2}$$

Existence of two conserved quantities



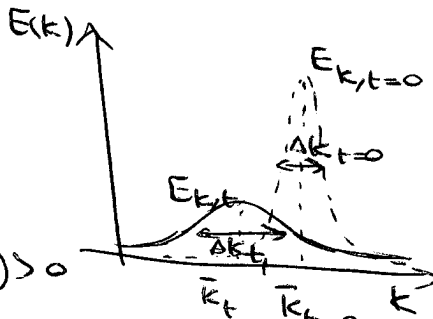
Theory of dual-cascade (Kraichnan, 67):

- Self-similar <sup>local</sup> eustrophy cascade toward high  $k$ 's.
- Self-similar local energy flux cascade toward low  $k$ 's (inverse cascade).

Inverse Cascade??

Turbulent act toward

broadening  $\Delta k^2$ :  $\partial_t \langle \Delta k^2 \rangle > 0$



$\Delta k$ : variance of distribution  
 $\bar{k}$ : centroid of distribution

$$\Delta k^2 = \frac{\int dk (k - \bar{k})^2 E(k)}{\int dk E(k)} = \frac{\int dk (k^2 - 2k\bar{k} + \bar{k}) E(k)}{E_0}$$

$E_0$ : conserved energy

①:  $\int dk k^2 E(k) = \Omega_0$  conserved eustrophy

②:  $-2\bar{k} \frac{\int dk k E(k)}{E_0} = -\bar{k} \frac{E_0}{E_0}$

③:  $\int dk \bar{k}^2 E(k) = \bar{k}^2 E_0$

$$\Rightarrow \partial_t \langle \Delta k^2 \rangle = \partial_t \left( \frac{\Omega_0 - \bar{k}^2 E_0}{E_0} \right) = -\partial_t \bar{k}^2$$

$$\Rightarrow \partial_t \langle (\Delta k)^2 \rangle = - \partial_t k^{-2}$$

since this is > 0

$$\rightarrow \uparrow < 0$$

- Spectral Broadening  
- Downshift of Spectral centroid

↓  
Idea of inverse cascade!

Constructing spectra using scaling arguments:

⇒ Enstrophy Cascade:

$$kE(k) \sim \langle v^2 \rangle \Rightarrow \frac{1}{\tau_{\text{cascade}}} = \frac{1}{\tau_{\text{eddy turnover}}} = \frac{v(l)}{l} = k(kE(k))$$

Also  $k^3 E(k)$  corresponds to Enstrophy density:

$$\text{Enstrophy Dissipation Rate: } \eta = \frac{k^3 E(k)}{\tau_{\text{cascade}}}$$

$$\Rightarrow \eta = (k^3 E(k))^{3/2} \Rightarrow \boxed{E(k) = \eta^{2/3} k^{-3}}$$

Forward Cascade

⇒ Inverse Energy Cascade:

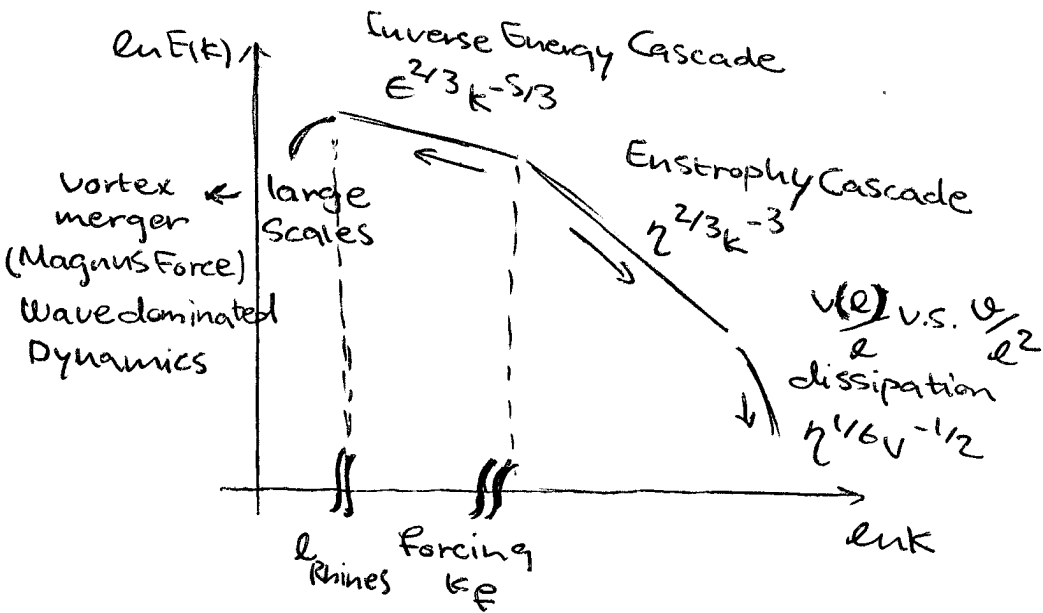
Balancing energy dissipation rate with

slow rate of energy to larger scales:

$$\epsilon = \frac{kE(k)}{\tau_{\text{cascade}}} \Rightarrow \epsilon = k^{5/2} E(k)^{3/2}$$

$$\Rightarrow \boxed{E(k) = \epsilon^{2/3} k^{-5/3}}$$

Inverse Cascade



Rossby Wave:

$$\partial_t \nabla^2 \phi + \nu \nabla^4 \phi = \beta \partial_x \phi + \mu \nabla^2 \nabla^2 \phi + \dots$$

$$\Rightarrow \omega_k = -\frac{\beta k_x}{k^2}$$

Strong dispersion at large Scales

Drag at large scales:

$$\partial_t w + \nu \nabla^2 w = -\mu w + \nu \nabla^2 w$$

↳ Drag → Bottom Ekman Effects  
↳ Toroidal Effects

Rhines length scale:

Cross-over Transition length scale in which eddy dominated dynamics will become wave interaction dynamics.

- Wave dominated:  $\omega_k T_c / k > 1$
- Eddy dominated:  $\omega_k T_c / k < 1$

$$\tilde{w} \text{ vs. } \frac{\beta k_x}{k^2} \Rightarrow k^2_{\text{cross-over}} = \frac{\beta}{\nu}$$

inverse cascade vs. ZF

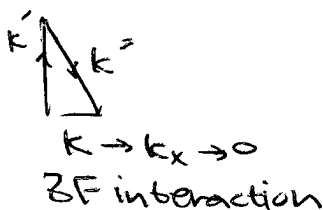
$$\Rightarrow l_{\text{Rhines}} = \sqrt{\frac{\nu}{\beta}}$$

Rhines Mechanism:

$$\pi \text{ wave} \sim \delta(\omega_k - \omega_{k'} - \omega_{k''})$$

↳ Two Rossby Waves + One Zonal Flow

- Drift-wave  $\partial_z F = \eta$
- Lack of clear self-similarity range
- Lack of dispersion at large Scales





Dispersion of particle pairs (Richardson's Problem) For 2D turbulence:

particle distance  $l$ :  $\frac{dl}{dt} = v(l)$

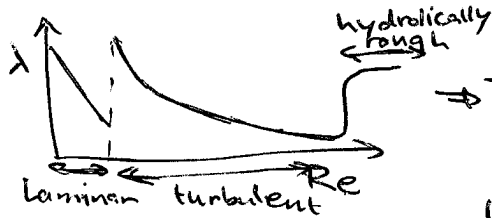
$l > k_g^{-1}$ :  $v(l) = \epsilon^{1/3} l^{1/3} \Rightarrow \boxed{l^2 \sim \epsilon t^3}$

Inverse Cascade  
(Same as k41)  
(grows super diffusively)

$l < k_g^{-1}$ :  $v(l) = (kE(k))^{1/2} \Rightarrow E(k) = \eta^{2/3} k^{-3}$

$\Rightarrow \frac{d}{dt} l = \eta^{1/3} l$  Forward Cascade  
(grows exponentially in time)

Poiseuille      Prandtl  
↑                    ↑  
Laminar vs turbulent;



⇒ Turbulence in Pipes & channels:

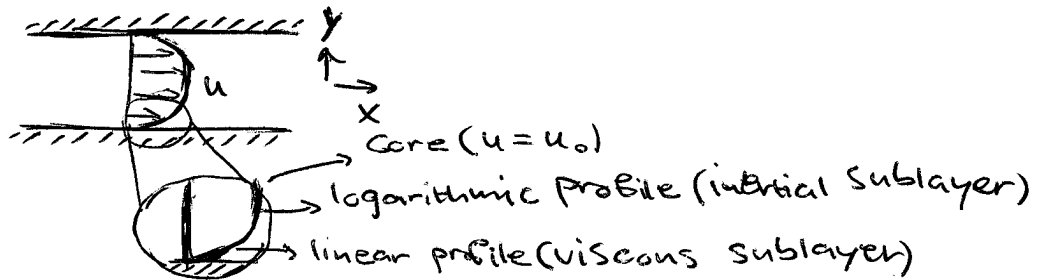
resistance:  $\lambda = 2a \frac{\Delta P / l}{\rho U^3}$   
Pressure drop  
mean flow energy  $\frac{1}{2} \rho U^2$

Up to now | - homogeneous flow in periodic box  
              | - cascade in scale space (Kolmogorov)

For pipes | - inhomogeneous flow  
              | - Momentum transport in a boundary layer (Prandtl)

Characteristics of logarithmic profile:

- profile consistency over wide Re Range
- Logarithmic profile is universal (Prandtl



"Law of the wall")

Similarity to k41: pipe flow turbulent manifests an element of universality.

- Turbulence resistance curve also universal!

Driven by  $\rightarrow$  turbulent mixing of cross-stream  
 Shear of mean flow by Reynolds  
 Stress.

Pipe flow  
 $\downarrow$   
 Simple example of  
 flux-driven turbulence  
 $\downarrow$   
 Also tokamaks,  
 Solar convections, etc

Turbulent Energy Production:

$$P = - \underbrace{\langle \tilde{v}_y \tilde{v}_x \rangle}_{\text{Reynolds Stress}} \frac{d}{dy} \underbrace{v_x(y)}_{\text{cross-stream shear}}$$

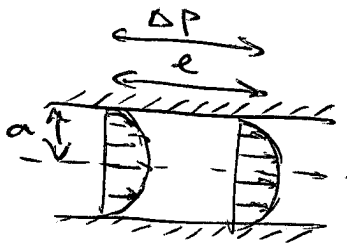
no-slip boundary  
 condition:

$\rightarrow$  Viscous Sublayer:

$$\lim_{x \rightarrow 0} u(x) \rightarrow 0$$

Momentum flux to wall  $\Rightarrow$  Stress on the wall  
 $\downarrow$

must balance pressure  
 drop ( $\Delta P$ )



$$\text{Force on wall} = \tau u_*^2 A_{\text{wall}}$$

$$\text{Force on fluid} = \Delta P A_{\text{flow}}$$

$u_*$ : friction velocity

$\tau u_*$ : wall stress

$\tau u_*^2 = \tau_f$ : momentum flux

$$\tau u_*^2 (2\pi a l) = \Delta P (\pi a^2) \Rightarrow u_* = \left[ \left( \frac{\Delta P}{2l} \right) \left( \frac{a}{2l} \right) \right]^{1/2}$$

characteristic velocity for  
 pipe flow  $\leftarrow$

In viscous sublayer

$$Re = u_* y < 1$$

satisfies relation  
 ( $Re < 1$ )

$\&$

$$u(y) \sim u_* \frac{y}{y_d}$$

Viscous sublayer width:  $y_d = \nu u_*^{-1}$

characteristic scale  
 for pipe flow  $\leftarrow$

$\rightarrow$  Inertial Sublayer (log law of the wall):

Assumption: scale invariance on  $y_d \ll y < a$   
 $\downarrow$   
 empirically!

Flow gradient is independent in inertial

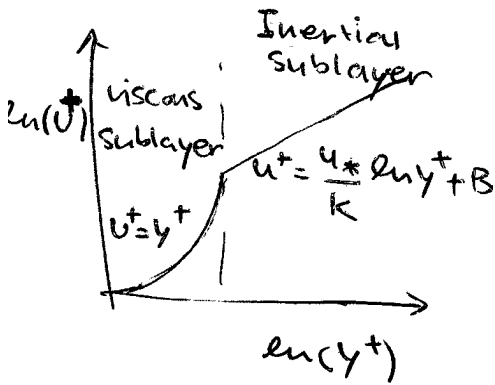
Scale:

$$\frac{du(y)}{dy} \sim \frac{u_*}{y}$$

$$\Rightarrow u(y) = \frac{u_*}{k} \ln(y/y_d) + \text{const.}$$

Log law of the wall

$k \approx 0.4$  constant  $\leftarrow$   
 von-karman constant (universal constant)



$$u^+ = \frac{u(y)}{u_*}; \quad y^+ = \frac{y}{y_d}$$

$$\langle \tilde{u} \tilde{v} \rangle = -\nu_T \partial_y u$$

$$\Rightarrow \tau = -(\nu + \nu_T) \partial_y u$$

Reynolds Averaged Navier-Stokes (RANS):

Shear:  $\tau = -\langle \tilde{u} \tilde{v} \rangle + \nu \partial_y u$   $\begin{cases} u = v_y \\ v = v_x \end{cases}$

Momentum in x:  $\frac{1}{\rho} \partial_x P = -\partial_y \langle \tilde{u} \tilde{v} \rangle + \nu \partial_{yy} u$   
 $= -\partial_y \tau$

$$\Rightarrow \left(\frac{1}{\rho} \partial_x P\right) y = -\langle \tilde{u} \tilde{v} \rangle + \nu \partial_y u + \text{const.}$$

B.C.:  $\partial_y u|_{y=0} = -\frac{u_*^2}{\nu}$

B.C.  $y=a \Rightarrow$  Statistically symmetric  $\Rightarrow$

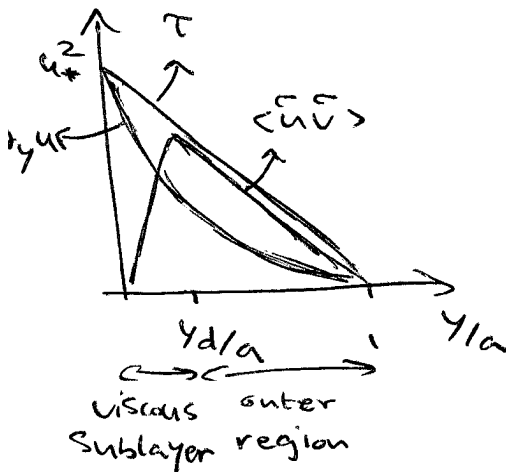
$$\langle \tilde{u} \tilde{v} \rangle|_y = -\langle \tilde{u} \tilde{v} \rangle|_{2a-y} \Rightarrow \langle \tilde{u} \tilde{v} \rangle|_a = 0$$

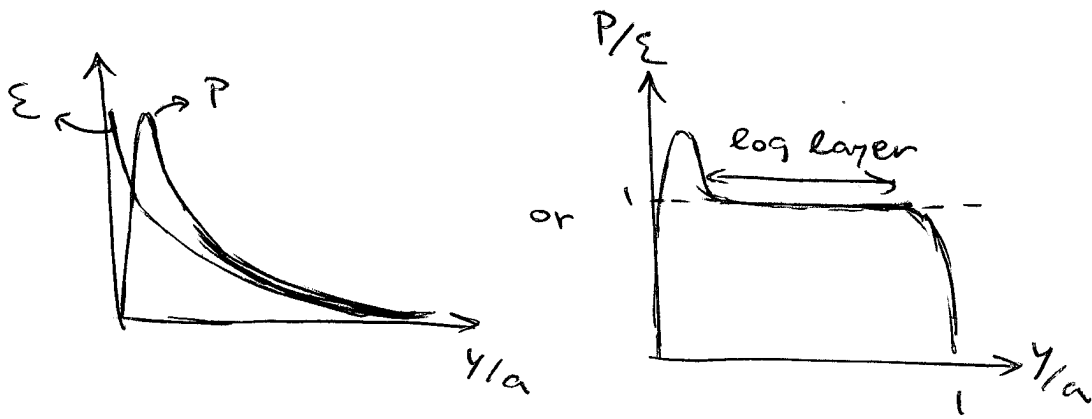
$$\Rightarrow \tau = -\langle \tilde{u} \tilde{v} \rangle + \nu \partial_y u = u_*^2 \left(1 - \frac{y}{a}\right)$$

Now if are calculating energy dissipation:

$$\xi = \nu \int \partial_y^2 u|^2$$

We will have:





We have turbulence energy as:

$$\frac{\partial E}{\partial t} = P - \epsilon = u_*^2 \left( \frac{u_*^2}{y^2} \right) - \frac{u_*^3}{l}$$

$l$ : characteristic  
lengthscale of  
turbulence

turbulent viscosity  $\propto \frac{\rho}{\rho} \left( \frac{\partial u(y)}{\partial y} \right)^2$  Note that:  $\frac{\tau_w}{\rho} = \langle \overline{u_y u_x} \rangle = \nu_T \frac{\partial u}{\partial y}$

stationary sublayer  $\Rightarrow l \sim y$

turbulent lengthscale in distance  
from wall!

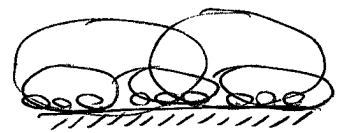
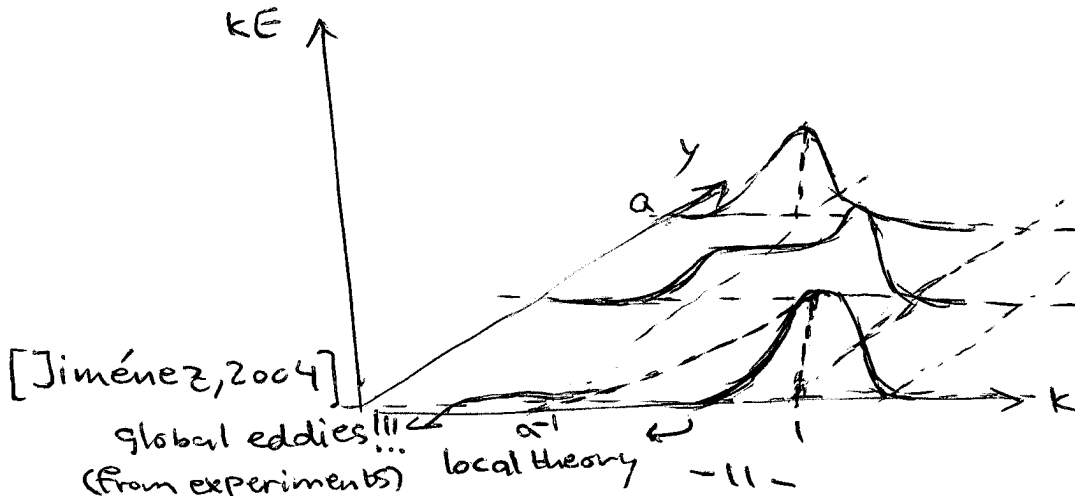
$$\Rightarrow \nu_T = u_* y$$

eddy viscosity  
or turbulent viscosity

"Mixing length  
theory always  
works... Provide  
you know the  
mixing length...  
- P.D.

rate of turbulent transport of momentum

Spectra for wall turbulence:



observation shows  
local theory does  
work!  
Non-locality?

→ Approach to self-similarity:

Again, we can formulate the problem "mean velocity gradient" which is locally self-similar and seemingly "universal".

- Dimensionless function  $y u_*^{-1} \frac{\partial u(y)}{\partial y}$  is determined by dimensionless parameters in problem:

$$\frac{y}{u_*} \frac{\partial u(y)}{\partial y} = F\left(\frac{y_d}{y}, \frac{y}{a}\right)$$

In inertial sublayer and high Re

↓

$$y/y_d \gg 1 \quad \& \quad y \ll a$$

assuming complete  
Re number similarity ↓

$$y_d/y \rightarrow 0 \quad \& \quad y/a \rightarrow 0$$

$$\Rightarrow \frac{y}{u_*} \frac{\partial u(y)}{\partial y} = F(0, 0) \rightarrow \text{const.}$$

↑

log law of the wall

Additional  
References:

[Pope, Turbulent  
Flows]

[Tennekes & Lumley,

A First course in Turbulence]

→ Conclusion and practical issues:

The parallel between Prandtl and K41

has been mentioned before. See

below for details of comparison:

	Inertial Range Spectrum (K41)	Pipe Flow turbulence (Prandtl)
Self-similarity	in scale inertial range Spectrum $v(l)$	in space inertial sublayer profile $v(y)$
Scales ↓ invariance	$l_0, l_n, l_d$ $l \rightarrow$ scale space	$a, y, u/u_*$ $y \rightarrow$ real space
layers	{ inertial range dissipation range	{ inertial sublayer viscous sublayer
Throughput	$G = \frac{v(l)^2}{\tau(l)}$	$u_*^2 = \nu_T \frac{\partial u}{\partial y}$
Rate	$1/\tau(l) \sim v(l)/l$ (eddy turnover)	$\nu_T y^{-2} \sim u_*/y$
Balance	$v(l) \sim \epsilon^{1/3} l^{1/3}$	$\partial u(y)/\partial y \sim u_*/y$ (log profile)
Universality	universal spectral scaling	universal profile
Dissipation lengthscale	$l_d = \nu^{3/4} G^{-1/4}$	$y_d = \nu u_*^{-1}$
Fit constant	Kolmogorov constant	Voh-Karman constant

In this case, Prandtl attacked problem better!

→ Practical Issues:

- Parallel between Prandtl and k41 doesn't exist in dealing with rigorous results.

triple moment:  $\begin{cases} k41: \langle \delta v^3(r) \rangle = -\frac{4}{5} \epsilon r & \boxed{\frac{4}{5} \text{ law}} \\ P32: \langle \delta v^3(r) \rangle \approx u_*^3 = \epsilon r \end{cases}$

↓  
No theory available at this time!

- Resistance law v.s. Pipe flows

$$y_d < y < a \xrightarrow{y \rightarrow a} u \approx \frac{u_*}{\kappa} \ln\left(\frac{u_* a}{\nu}\right) *$$

$$u_* = \left(\frac{a \Delta P}{\rho \kappa^2}\right)^{1/2} \Rightarrow u = \left(\frac{a \Delta P}{2 \rho \kappa^2}\right)^{1/2} \ln\left(a \frac{a \Delta P}{2 \rho \nu}\right)$$

using  $Re = \frac{2au}{\nu}$  and  $\lambda = \frac{2a \Delta P / \rho}{\frac{1}{2} \rho u^2}$  we

can rewrite friction law as:

$$\begin{cases} 1/\sqrt{\lambda} = .88 \ln(Re \sqrt{\lambda}) - .85 \\ Re = 2au/\nu; \lambda = (2a \Delta P / \rho) / (\frac{1}{2} \rho u^2) \end{cases}$$

good fit to pipe flow data??

## References:

- [1] Diamond, P.H., Itoh, S., Itoh, K., "Modern Plasma Physics" (2007)
- [2] PHYS 218C Lecture Notes, by Prof. Diamond
- [3] MAE 214A Lecture Notes, by Prof. DelAlamo
- [4] Jiménez, J., "Turbulence and vortex dynamics" (2004)
- [5] Pope, S.B., "Turbulent Flows" (2000)